1. The theoretical and calculated values for stress and strain for the plate extension problem all were equal.

	σх	σу	τ _{xy}	$\mathbf{\epsilon}_{\mathbf{x}}$	$\mathbf{\epsilon}_{\mathrm{y}}$	γ_{xy}
Theoretical	14 MPa	0	0	200 με	-70 με	0
Calculated	14 MPa	0	0	200 με	70 με	0

- 2. The stress profile for the analytical solution for the straight beam is similar to the stress profile for the radial stress for the caliper for the cross-section. The locations of the neutral axes do not differ. Meanwhile, the first principal stress approximates the longitudinal stress for negative values from the neutral axis while it approximates the radial stress for positive values. The opposite is true for the second principal stress. Both are true for the coarse and fine meshes.
- 3. The stress and strain calculations are not affected by the approximate locations of the modeling of the brake geometry; they are determined by the displacements at each node.













